

- All integration schemes are only *conditionally stable*
 - I.e.: they are only stable for a specific range for Δt
 - This range depends on the stiffness of the springs
- Goal: *unconditionally stability*
- One option: **implicit Euler integration**

explicit

$$\mathbf{x}_i^{t+1} = \mathbf{x}_i^t + \Delta t \mathbf{v}_i^t$$

$$\mathbf{v}_i^{t+1} = \mathbf{v}_i^t + \Delta t \frac{1}{m_i} \mathbf{f}(\mathbf{x}^t)$$

implicit

$$\mathbf{x}_i^{t+1} = \mathbf{x}_i^t + \Delta t \mathbf{v}_i^{t+1}$$

$$\mathbf{v}_i^{t+1} = \mathbf{v}_i^t + \Delta t \frac{1}{m_i} \mathbf{f}(\mathbf{x}^{t+1})$$

- Now we've got a system of non-linear, algebraic equations, with \mathbf{x}^{t+1} and \mathbf{v}^{t+1} as unknowns on **both** sides → **implicit integration**

- Write all the implicit equations as **one big** system of equations :

$$M\mathbf{v}^{t+1} = M\mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^{t+1}) \quad (1)$$

$$\mathbf{x}^{t+1} = \mathbf{x}^t + \Delta t \mathbf{v}^{t+1} \quad (2)$$

- Plug (2) into (1) :

$$M\mathbf{v}^{t+1} = M\mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t + \Delta t \mathbf{v}^{t+1}) \quad (3)$$

- Expand \mathbf{f} as Taylor series:

$$\begin{aligned} \mathbf{f}(\mathbf{x}^t + \Delta t \mathbf{v}^{t+1}) = & \mathbf{f}(\mathbf{x}^t) + \frac{\partial}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}^t) \cdot (\Delta t \mathbf{v}^{t+1}) \\ & + O((\Delta t \mathbf{v}^{t+1})^2) \end{aligned} \quad (4)$$

- Plug (4) into (3):

$$\begin{aligned}
 M \mathbf{v}^{t+1} &= M \mathbf{v}^t + \Delta t \left(\mathbf{f}(\mathbf{x}^t) + \underbrace{\frac{\partial}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}^t)}_K \cdot (\Delta t \mathbf{v}^{t+1}) \right) \\
 &= M \mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t) + \Delta t^2 K \mathbf{v}^{t+1}
 \end{aligned}$$

- K is the Jacobi-Matrix, i.e., the derivative of \mathbf{f} (w.r.t.x):

$$K = \begin{pmatrix} \frac{\partial}{\partial x_{11}} f_{11} & \frac{\partial}{\partial x_{12}} f_{11} & \dots & \frac{\partial}{\partial x_{n3}} f_{11} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{\partial}{\partial x_{11}} f_{n3} & \dots & \dots & \frac{\partial}{\partial x_{n3}} f_{n3} \end{pmatrix}$$

- K is called the **tangent stiffness matrix**
 - (The normal stiffness matrix is evaluated at the equilibrium of the system: here the matrix is evaluated at an arbitrary "position" of the system in phase space, hence the name "*tangent ...*")

- Reorder terms :

$$(M - \Delta t^2 K) \mathbf{v}^{t+1} = M \mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t)$$

- Now, this has the form:

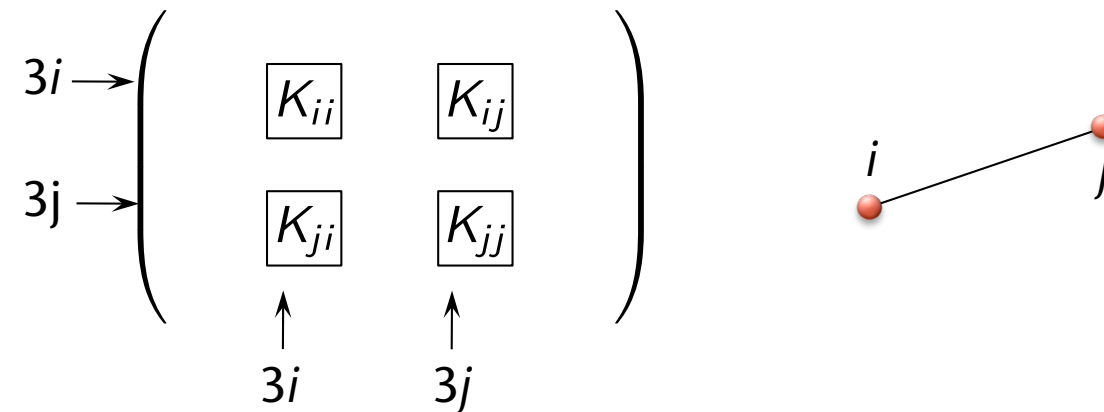
$$A \mathbf{v}^{t+1} = \mathbf{b}$$

$$\text{mit } A \in \mathbb{R}^{3n \times 3n}, \quad b \in \mathbb{R}^{3n}$$

- Solve this system of linear equations with any of the iterative solvers
- Don't use a non-iterative solver, because
 - A changes with every frame (simulation step)
 - We can "warm start" the iterative solver with the solution as of last frame

Computation of the Stiffness Matrix

- First, understand the anatomy of matrix K :
 - A spring (i, j) adds the following four 3×3 block matrices to K :



- Matrix K_{ij} arises from the derivation of $\mathbf{f}_i = (f_{i1}, f_{i2}, f_{i3})$ w.r.t. $\mathbf{x}_j = (x_{j1}, x_{j2}, x_{j3})$:

$$K_{ij} = \begin{pmatrix} \frac{\partial}{\partial x_{j1}} f_{i1} & \frac{\partial}{\partial x_{j2}} f_{i1} & \frac{\partial}{\partial x_{j3}} f_{i1} \\ \vdots & \vdots & \vdots \\ \frac{\partial}{\partial x_{j1}} f_{i3} & \dots & \frac{\partial}{\partial x_{j3}} f_{i3} \end{pmatrix}$$

- In the following, consider only f^s (spring force)

- First of all, compute K_{ij} :

$$\begin{aligned}
 K_{ij} &= \frac{\partial}{\partial \mathbf{x}_i} f_i(\mathbf{x}_i, \mathbf{x}_j) \\
 &= k_s \frac{\partial}{\partial \mathbf{x}_i} \left((\mathbf{x}_j - \mathbf{x}_i) - l_0 \frac{\mathbf{x}_j - \mathbf{x}_i}{\|\mathbf{x}_j - \mathbf{x}_i\|} \right) \\
 &= k_s \left(-I - l_0 \frac{-I \cdot \|\mathbf{x}_j - \mathbf{x}_i\| - (\mathbf{x}_j - \mathbf{x}_i) \cdot 2 \frac{(\mathbf{x}_j - \mathbf{x}_i)^\top}{\|\mathbf{x}_j - \mathbf{x}_i\|}}{\|\mathbf{x}_j - \mathbf{x}_i\|^2} \right) \\
 &= k_s \left(-I + l_0 \frac{1}{\|\mathbf{x}_j - \mathbf{x}_i\|} I + \frac{2l_0}{\|\mathbf{x}_j - \mathbf{x}_i\|^3} (\mathbf{x}_j - \mathbf{x}_i)(\mathbf{x}_j - \mathbf{x}_i)^\top \right)
 \end{aligned}$$

—

■ Zur Erinnerung:

■
$$\left(\frac{f}{g}\right)' = \frac{f'g - fg'}{g^2}$$

■
$$\frac{\partial}{\partial \mathbf{x}} \|\mathbf{x}\| = \frac{\partial}{\partial \mathbf{x}} \left(\sqrt{x_1^2 + x_2^2 + x_3^2} \right) = 2 \frac{\mathbf{x}^T}{\|\mathbf{x}\|}$$

- Aus einigen Symmetrien folgt:

- $K_{ij} = \frac{\partial}{\partial \mathbf{x}_j} f_i(\mathbf{x}_i, \mathbf{x}_j) = -K_{ii}$

- $K_{jj} = \frac{\partial}{\partial \mathbf{x}_j} f_j(\mathbf{x}_i, \mathbf{x}_j) = \frac{\partial}{\partial \mathbf{x}_j} (-\mathbf{f}_i(\mathbf{x}_i, \mathbf{x}_j)) = K_{ii}$

- $K_{ji} = K_{ij}$

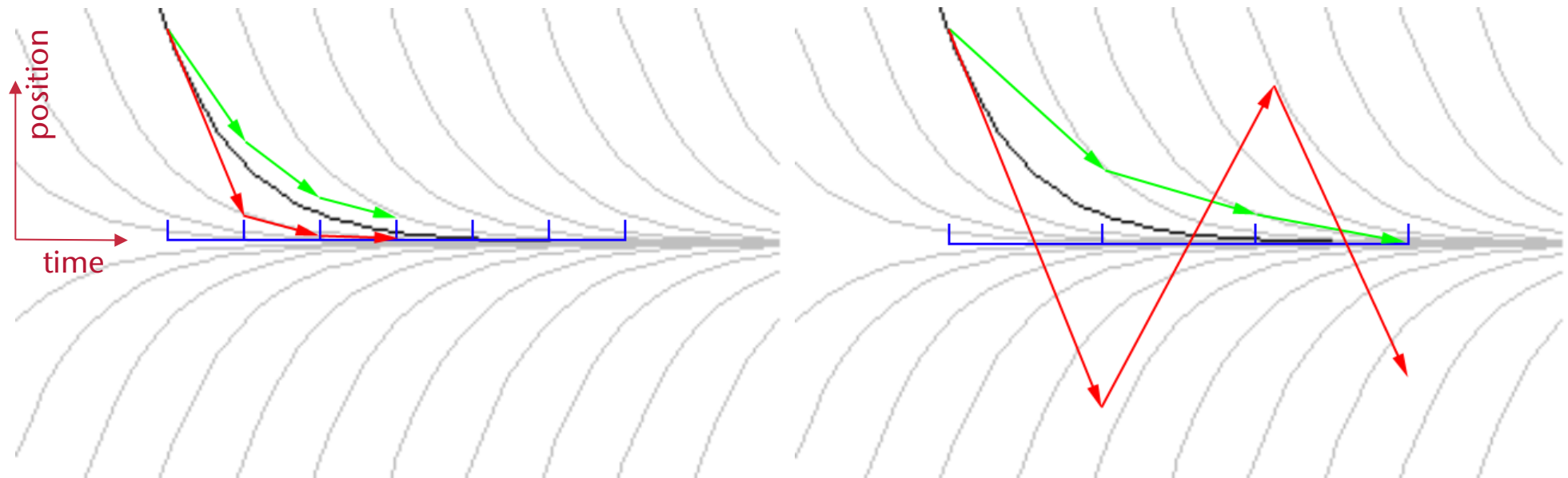
Overall Solution Algorithm

- Initialize $K = 0$
- For each spring (i, j) compute $K_{ii}, K_{ij}, K_{ji}, K_{jj}$ and accumulate it to K at the right places
- Compute $\mathbf{b} = M \mathbf{v}^t + \Delta t f(\mathbf{x}^t)$
- Solve the linear equation system $A \mathbf{v}^{t+1} = \mathbf{b} \rightarrow \mathbf{v}^{t+1}$
- Compute $\mathbf{x}^{t+1} = \mathbf{x}^t + \Delta t \mathbf{v}^{t+1}$

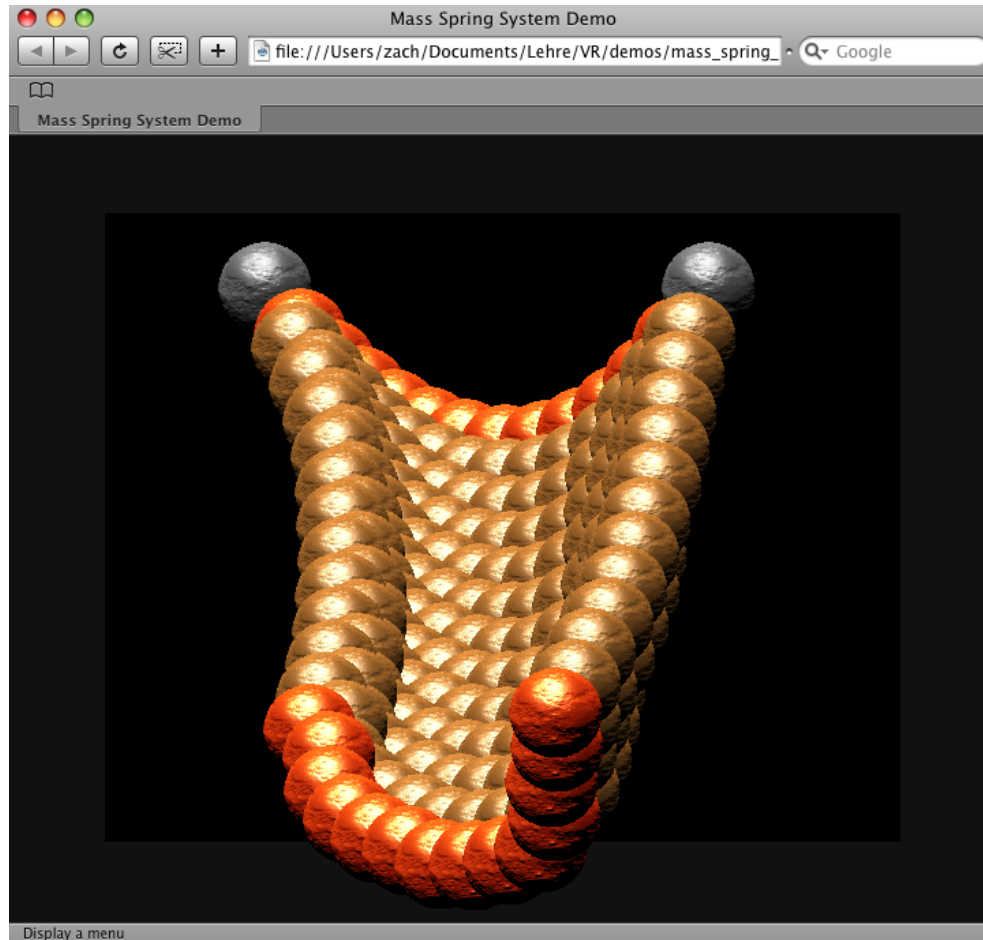
Advantages and Disadvantages

- **Explicit** integration:
 - + Very easy to implement
 - Small step sizes needed
 - Stiff springs don't work very well
 - Forces are propagated only by one spring per time step
- **Implicit** Integration:
 - + Unconditionally stable
 - + Stiff springs work better
 - + Globale solver → forces are being propagated throughout the whole spring-mass system with one time step
 - Large time steps are needed, because one step is much more expensive (if real-time is needed)
 - The integration scheme introduces damping by itself (might be unwanted)

- Visualization of: $\dot{x}(t) = -x(t)$

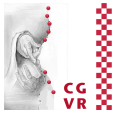


- Informal Description:
 - Explicit** jumps forward behindly, based on current information
 - Implicit** jumps backward and tries to find a future position such that the backwards jump arrives exactly at the current point (in phase space)



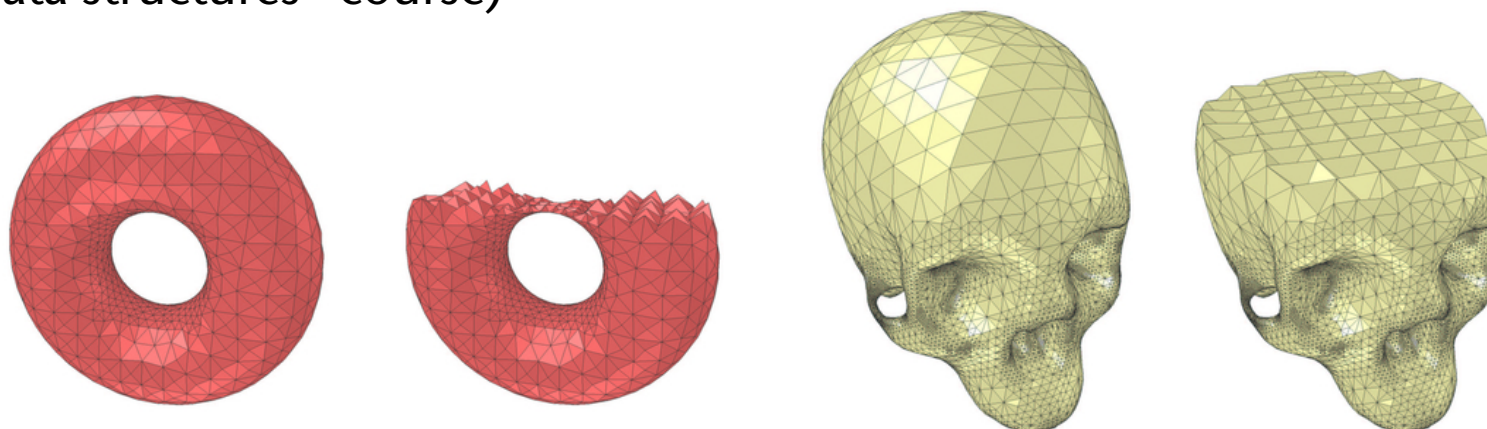
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Mesh Creation for Volumetric Objects



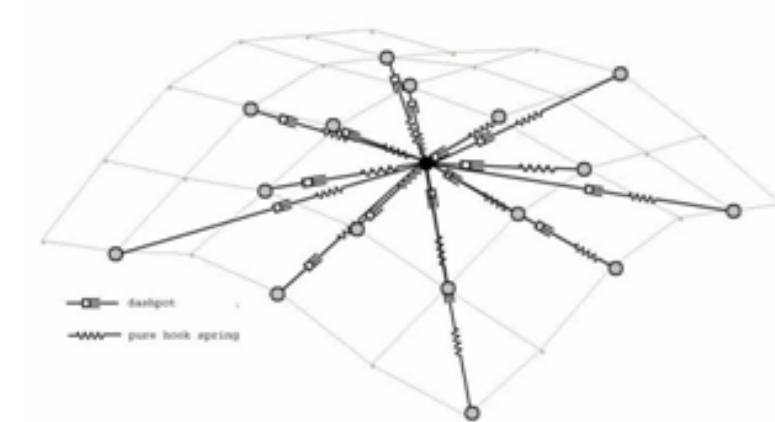
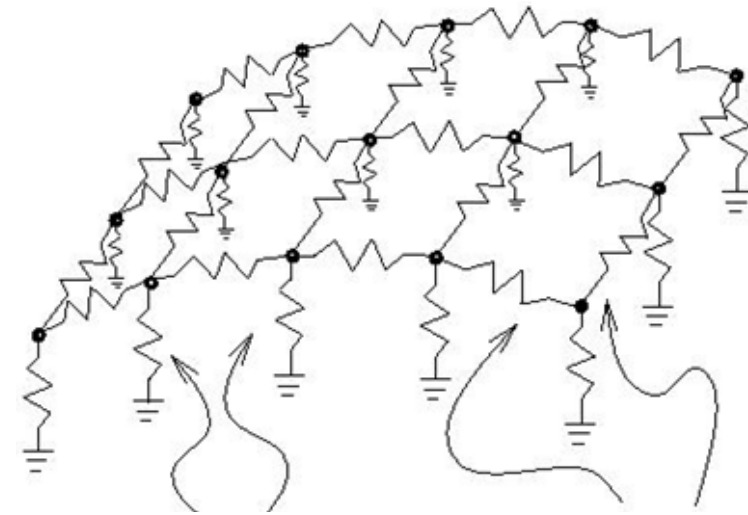
- How to create a mass-spring system for a **volumetric** model?
- Direct conversion of 3D (surface) geometry into spring-mass system does not yield good results:
 - Geometry has too high a complexity
 - Degenerate polygons
- Better (and still simple) idea:
 - Create a tetrahedron mesh out of the geometry (somehow)
 - Each vertex (node) of the tetrahedron mesh becomes a mass point, each edge a spring
 - Distribute the masses of the tetraeder (= density \times volume) equally among the mass point

- Generation of the tetrahedron mesh (simple method):
 - Distribute a number of points uniformly (perhaps randomly) in the interior of the geometry (so called "**Steiner points**")
 - Dito for a sheet/band above the surface
 - Connect the points by Delaunay triangulation (see my "Geometric Data structures" course)

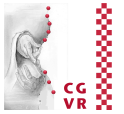


- Anchor the surface meshes within the tetraeder mesh:
 - Represent each vertex of the surface mesh by barycentric combination of ist surrounding tetraeder mesh

- In addition (optionally):
 - Anchor the outer mass points (of the tetrahedron mesh) at (imaginary) walls
 - Introduce diagonal "struts" (Streben)



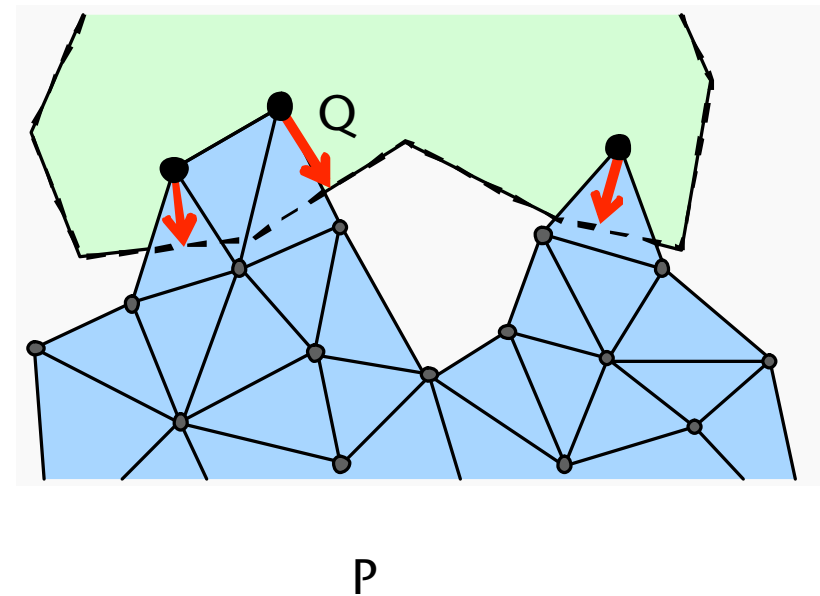
Collision Detection



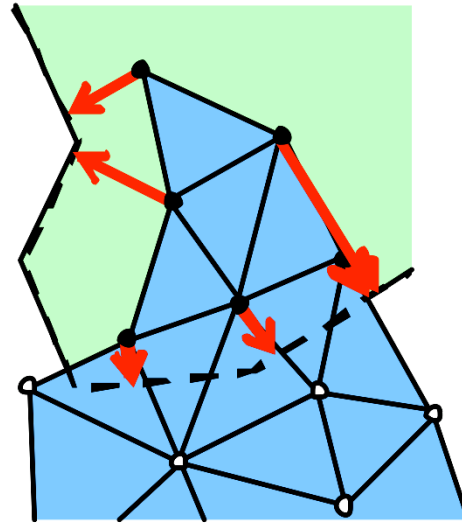
- Put all tetrahedra in a 3D grid (use a hash table!)
- In case of a collision in the hash table:
 - Compute exact intersection between the 2 involved tetrahedra

Collision Response

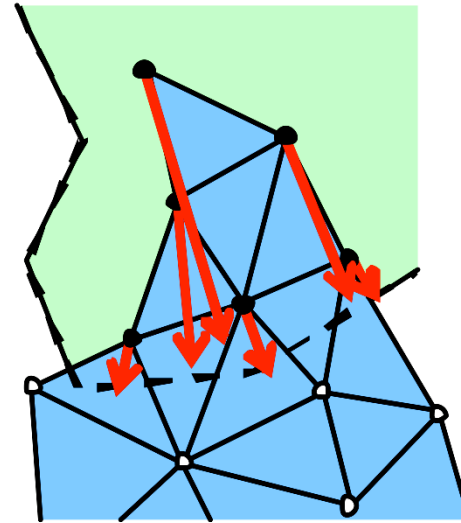
- Task: objects P and Q (= tetrahedral meshes) collide — what is the **penalty force**?
- Naïve approach:
 - For each mass point of P that has penetrated, compute its closest distance from the surface of Q → force (amount + direction)
- Problem:
 - Implausible forces
 - "Tunneling" (s. a. the chapter on force-feedback)



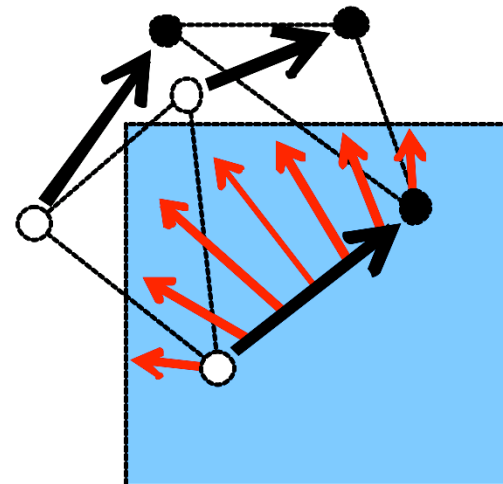
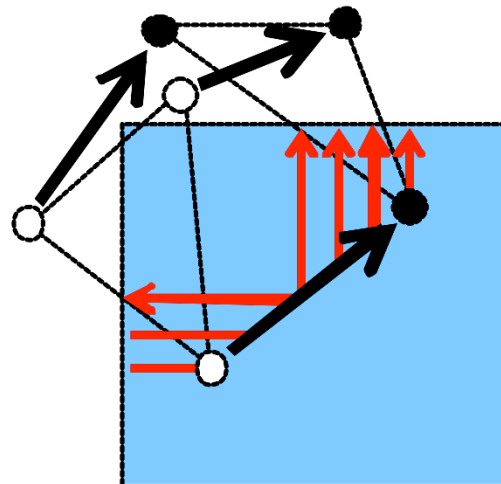
■ Examples:



inconsistent

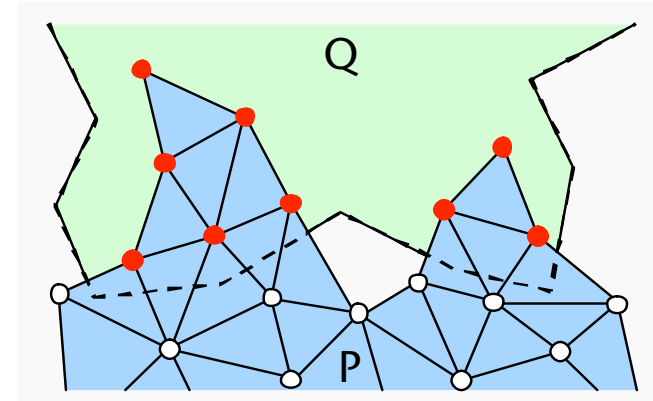


consistent



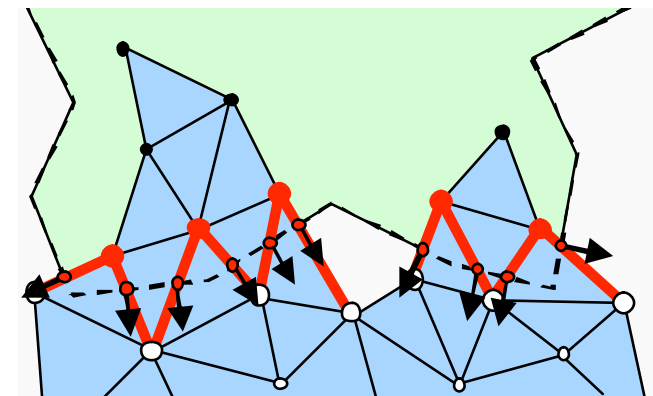
Consistente Penalty Forces

1. Phase: identify all points of P that penetrate Q



2. Phase: determine all edges of P that intersect the surface of Q

- For each such edge, compute the exact intersection point x_i
- For each intersection point, compute a normal n_i
 - E.g., by barycentric interpolation of the vertex normals of Q



3. Phase: compute the approximate force for border points

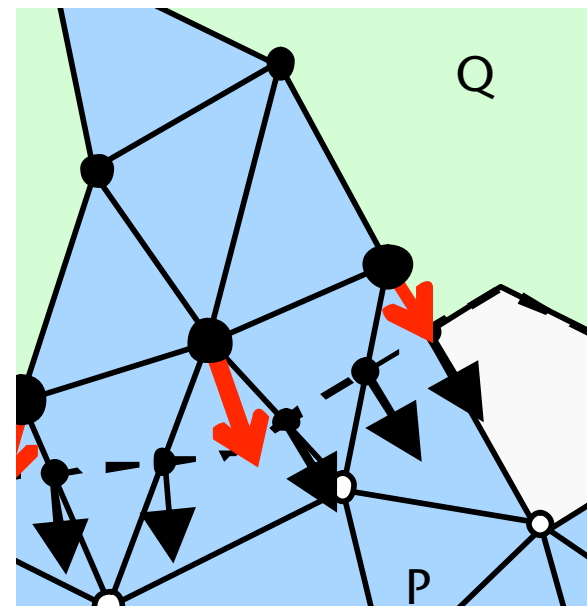
- Border point = a point \mathbf{p} that penetrates Q and is incident to an intersecting edge
- Observation: a border point can be incident to several intersecting edges
- Set the penetration depth for point \mathbf{p}

to

$$d(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p}) (\mathbf{x}_i - \mathbf{p}) \cdot \mathbf{n}_i}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

where $d(\mathbf{p})$ = approx. penetration depth of mass point \mathbf{p} , \mathbf{x}_i = point of the intersection of an edge incident to \mathbf{p} with surface Q , \mathbf{n}_i = normal to surface of Q at point \mathbf{x}_i ,

and $\omega(\mathbf{x}_i, \mathbf{p}) = \frac{1}{\|\mathbf{x}_i - \mathbf{p}\|}$



- Direction of the penalty force on border points:

$$\mathbf{r}(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p}) \mathbf{n}_i}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

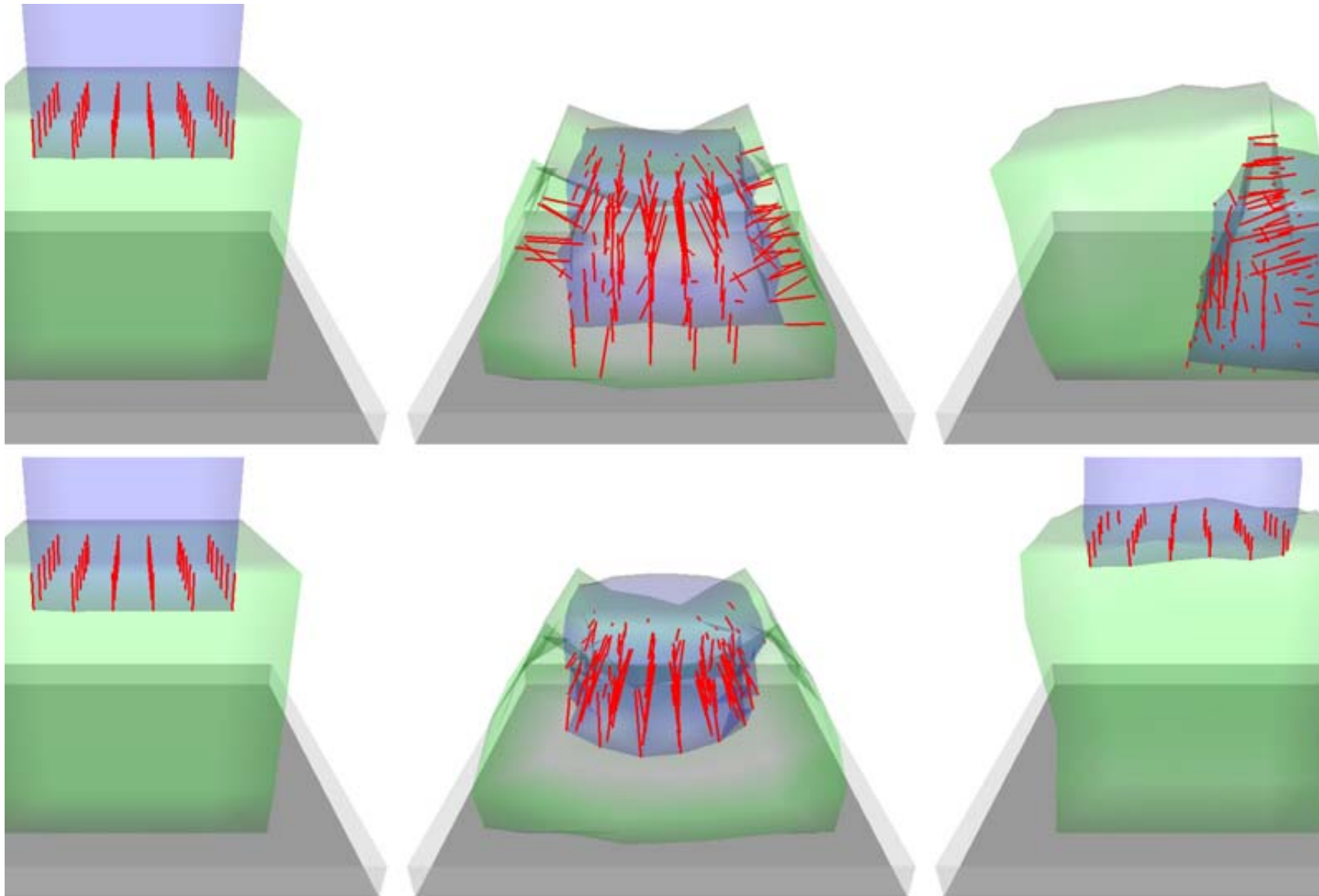
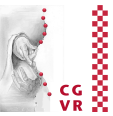
4. Phase: propagate forces by way of breadth-first traversal through the tetrahedron mesh

$$d(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{p}_i, \mathbf{p}) ((\mathbf{p}_i - \mathbf{p}) \cdot \mathbf{r}_i + d(\mathbf{p}_i))}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

where \mathbf{p}_i = points of P that have been visited already, \mathbf{p} = point not yet visited, \mathbf{r}_i = direction of the estimated penalty force in point \mathbf{p}_i .



Visualization



Consistent Penetration Depth Estimation for Deformable Collision Response

<http://cg.informatik.uni-freiburg.de>